

RESERVE DESK

Dynamics & Vibrations Ph.D.
Qualifier Exam
Fall Quarter 1995 - Page 1

5

DEC 20 1995

GEORGIA INSTITUTE OF TECHNOLOGY

The George W. Woodruff
School of Mechanical Engineering

Ph.D. Qualifiers Exam - Fall Quarter 1995

Dynamics & Vibrations

EXAM AREA

Assigned Number **(DO NOT SIGN YOUR NAME)**

-- Please sign your name on the back of this page --

Dynamics and Vibrations Qualifying Exam, Fall, 1995

Work any 3 of the 4 problems. Show all of your work. Clearly state all assumptions.

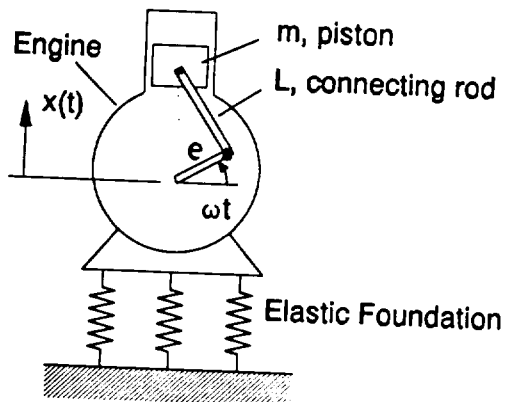
A one-cylinder reciprocating engine rests on a lossless elastic foundation as shown. The total mass of the engine is M while the mass of the piston alone is m . The elastic stiffness of the foundation is k . Assume that the crankarm is massless with length e and that the connecting rod is massless with length L .

- (a) Considering only the vertical motion and forces for this system, derive the equation of motion for the engine displacement $x(t)$. If possible, simplify the equation of motion assuming that $(e/L) \ll 1$.
- (b) Even if you could not complete part (a), consider the following approximate linear model for the engine:

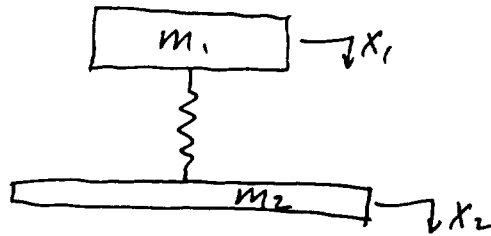
$$M \ddot{x} + k x = m e \omega^2 \left[\sin(\omega t) - \frac{e}{L} \cos(2\omega t) \right]$$

Find the steady-state response of the engine for the case $\omega = 2\omega_n$.

- (c) In terms of ω_n , what are the systems's critical rotation speeds?



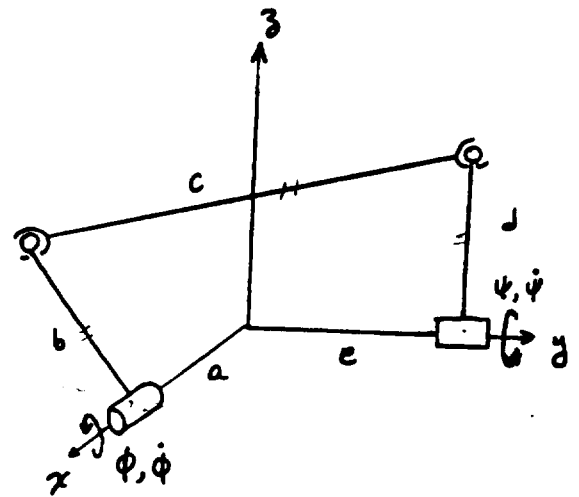
The system below represents a model for a device supported on a flexible foundation. The foundation is represented as a free mass.



- 1) Find the natural frequencies of this system.
- 2) Assume that a disturbing force $f(t) = P_0 \sin(\omega t)$ is applied to mass m_1 , and derive expressions for the motion of the foundation (m_2), and for the force transmitted to the foundation.
- 3) For vibration isolation, a general guide is that the natural frequency of the system on its support should be at most one-third the disturbance frequency. Suppose the device, m_1 , weighs 10 times its foundation, m_2 . For a fixed disturbance frequency ω_0 , compare the required natural frequency for vibration isolation of this flexible foundation model to the required natural frequency if the foundation were rigid and fixed in space.

The figure shows a 4-bar spatial RSSR* mechanism. Two revolute joints rotate links b and d that connect to link c by ball and socket joints. Angles ψ and ϕ are assumed known and are measured from the x-y plane using a right-hand rotation. Given the link lengths a, b, ... e find:

- The degrees of freedom of the linkage.
- $\dot{\phi}$ as a function of $\dot{\psi}$.
- The mechanical advantage for input ψ and output ϕ .



*RSSR - Revolute - Spherical - Spherical - Revolute

A rigid wheel has principal moments of inertia $I_1 = I_2 \neq I_3$ about its body-fixed principal axes \hat{x}_1, \hat{x}_2 and \hat{x}_3 , as shown in Fig. 1.179. The wheel is attached at its center of mass to a bearing which allows frictionless rotation about one space-fixed axis. The wheel is "dynamically balanced", i.e. it can rotate at constant $\omega \neq 0$ and exert no torque on its bearing. What conditions must the components of ω satisfy? Sketch the permitted motion(s).

