

RESERVE DESK

TRIBOLOGY QUALIFIER EXAM  
Spring 1995 - Page 1

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GEORGIA INSTITUTE OF TECHNOLOGY

The George W. Woodruff  
School of Mechanical Engineering

**Ph.D. Qualifiers Exam - Spring Quarter 1995**

TRIBOLOGY

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EXAM AREA

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Assigned Number (**DO NOT SIGN YOUR NAME**)

-- Please sign your name on the back of this page --

**GEORGE W. WOODRUFF SCHOOL OF MECHANICAL ENGINEERING**  
**Ph.D. QUALIFYING EXAM-TRIBOLOGY**  
**SPRING, 1995**

**QUESTION NO. 1**

In Hertzian contact analysis, we have, for point contact:

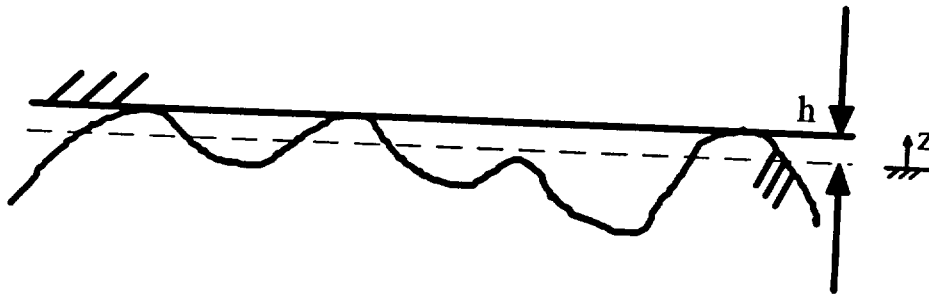
$$a = \left( \frac{3WR}{4E'} \right)^{\frac{1}{3}} \quad \delta = \frac{a^2}{R} \quad \frac{1}{R} = \frac{1}{R_1} + \frac{1}{R_2}$$

$$\frac{1}{E'} = \frac{1-\nu_1^2}{E_1} + \frac{1-\nu_2^2}{E_2}$$

where,

- $a$  = contact area half-width
- $R$  = reduced radius of curvature
- $\delta$  = interference = deformation at center of contact
- $W$  = load
- $E'$  = reduced modulus

Suppose an interface consists of a rough surface with rms roughness,  $\sigma$ , in contact with a smooth, rigid flat. Suppose also that the rough surface has an asperity height distribution,  $\phi(z)$ , and that there are  $\eta$  asperities per unit area. With reference to the figure below, derive an (integral) expression for the relationship between load and surface separation,  $h$ , as measured from the asperity mean height.



## QUESTION NO. 2

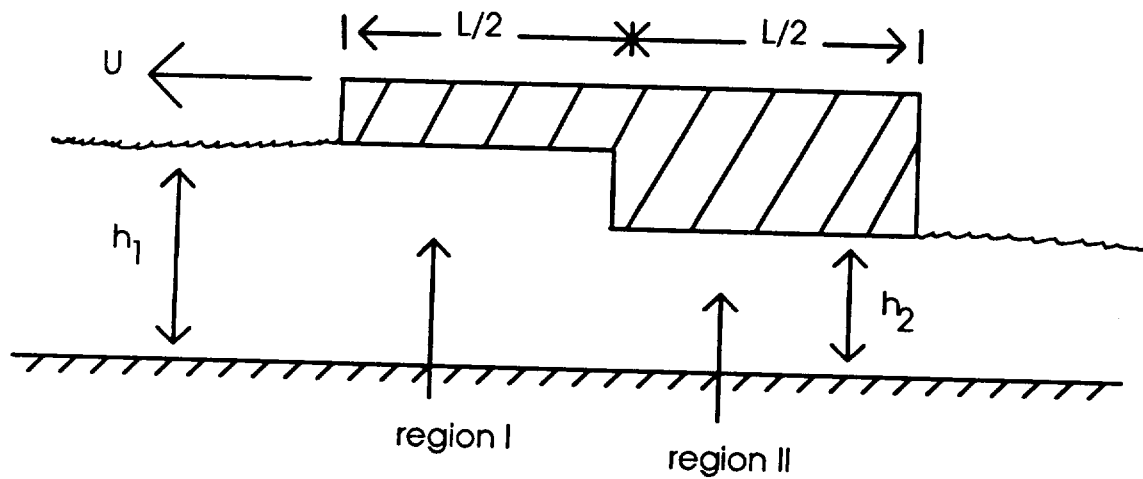
You are to design an hydrodynamic air-journal bearing for a high-speed dental drill (100,000 rpm). Under steady-state operation, the governing eqn. is given by:

$$\frac{\partial}{\partial x} \left( ph^3 \frac{\partial p}{\partial x} \right) + \frac{\partial}{\partial y} \left( ph^3 \frac{\partial p}{\partial y} \right) = 6\mu U \frac{\partial(ph)}{\partial x}$$

For very high rotational speeds, it is often argued that  $ph = \text{const}$  (approx.).

- (a) Justify this approximation based on the equation.
- (b) What kinds of loads do you expect for such a device in practice?
- (c) Suppose the journal is 6 mm in diameter and has a 6-mm axial length. Estimate the mean pressure that the bearing must generate in order to support the loads that you suggested above.
- (d) Given the approximation above, estimate the clearance required to support the loads.

**QUESTION NO. 3**



Consider the 2-dimensional step slider bearing shown above. The slider moves with speed  $U$  over a liquid film of constant density  $\rho$ , and constant viscosity  $\mu$ .

- Find the velocity profile in the fluid film in regions I and II
- Find the pressure distribution in the film and sketch it
- Find the maximum pressure in the film as a function of the relevant parameters (e.g.  $U$ ,  $h_1$ ,  $h_2$  etc.)
- Find the load which can be supported by the slider as a function of the relevant parameters

**HINT:** The flow in each region of the film can be approximated by a combination of plane Poiseuille Flow and Couette Flow