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RESERVE DESK

GEORGIA INSTITUTE OF TECHNOLOGI

The George W. Woodruf
School of Mechanical Engineering

Thermodynamics
EXAM AREA

Assigned Number (DO NOT SIGN YOUR NAME)

Please sign your <u>name</u> on the back of the page—

Please **print** your name here.

The Exam Committee will get a copy of this exame and will not be ric in whose paper it is until it is graded.

1(a) Using the Gibbs Equations and the Maxwell Relations below, derive a general differential for mush the enthalpy in terms of the pressure and temperature. In your derivation, please lentify and explain each stresult should be of the form,

$$dh = \left(\frac{\partial h}{\partial T}\right)_{P} dT + \left(\frac{\partial h}{\partial P}\right)_{V} dP$$

- 1(b) The coefficient of dT in the preceding equation is better known by what \blacksquare and symbol
- 1(c) Using the equation derived above, show that the enthalpy of an ideal gas repends only on the tenap 1
- 1(d) Using the result from Parts 2 and 3, derive the formula for the enth by of an ideal gas w.tl. specific heat in terms of T and P.
- 1(e) Explain in physical terms why the enthalpy does not depend on the produce. Your explanation is reference to the internal energy as well.
- 1(f) Using the appropriate Gibbs equation derive the integral equation for the entropy of an ideal affunction of temperature and pressure. This equation is of the form,

$$s = s(T, P)$$

Why are the previous results important in this derivation?

Table 1. The Gibbs Equations

$$dU = TdS - PdV$$

$$dH = TdS + VdP$$

$$dA = -SdT - PdV$$

$$dG = -SdT + VdP$$

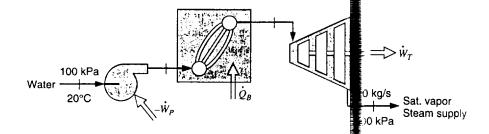
Table 2. The Maxwell Relations

$$\left[\frac{\partial T}{\partial V} \right]_{S,N_k} = -\left(\frac{\partial P}{\partial S} \right)_{V,N_k} \\
\left(\frac{\partial T}{\partial P} \right)_{S,N_k} = \left(\frac{\partial V}{\partial S} \right)_{P,N_k} \\
\left(\frac{\partial S}{\partial V} \right)_{T,N_k} = \left(\frac{\partial P}{\partial T} \right)_{V,N_k} \\
-\left(\frac{\partial S}{\partial P} \right)_{T,N_k} = \left(\frac{\partial V}{\partial T} \right)_{P,N_k}$$

2. A 10 kg/s steady supply of saturated-vapor steam at 500 kPa is required for drying a wood pull in a paper mill. You are to compare two alternative systems for roducing this saturat d vapor. One is a conventional system using a water pump to pump water to a steam generate (which exits the boiler at the required steam supply condition, i.e., 500 kPa, saturated vapor.

The second uses co-generation. In this system, the required steam steam ply will be the exhaust steam turbine as shown below. In this co-generation system, water 20C, 100 kPa, is pun pressure of 5 MPa and then fed to a steam generator (boiler) and to a steam turbine was exhausts at the required conditions. In both systems the isentropic and turbine are adiabation of the turbine is 85%. The pump and turbine are adiabation

- a) What is the difference in the heat transfer rate required by the stern generator between a systems?
- b) What is the difference in the net power between the two systems?
- by the co-generation system; i.e., the ratio of the additional net work alculated in b) divided additional heat required calculated in a)?



3. An adiabatic compressor is used to fill a 0.5 m³ rigid tank with at cospheric air (101 kP. Initially, the air in the tank is at 101 kPa, 25°C. The filling process the tank is isothermated the final pressure in the tank is 1000 kPa, what will be the minimum input work compressor?

- 4. A tank having a volume of 5 cubic feet contains initially nitrogen g at 11 atm and 70F develops a slow leak and, after a long time, the pressure drops to 5 and. Assume that the name an ideal gas. Assume the contents in the tank remain isothermal during the process.
- (a) Determine the mass of the nitrogen which escapes from the tank.
- (b) Determine the heat transfer, in BTU, through the tank walls.